FROM SOLAR PHOTOVOLTAIC COOLING FOR LIQUID DESICCANT REGENERATION IN A SOLAR AIR-CONDITIONING SYSTEM



FACULTY OF ENGINEERING UNIVERSITI MALAYSIA SABAH 2023

FROM SOLAR PHOTOVOLTAIC COOLING FOR LIQUID DESICCANT REGENERATION IN A SOLAR AIR-CONDITIONING SYSTEM

ZULKURNAIN BIN HASSAN

THESIS SUBMITTED IN FULFILMENT OF THE REQUIREMENTS FOR THE DEGREE OF MASTER OF ENGINEERING

FACULTY OF ENGINEERING UNIVERSITI MALAYSIA SABAH 2023

UNIVERSITI MALAYSIA SABAH

BORANG PENGESAHAN STATUS TESIS

JUDUL : EXPERIMENTAL STUDY ON USING WASTE HEAT FROM SOLAR

PHOTOVOLTAIC COOLING FOR LIQUID DESICCANT

REGENERATION IN A SOLAR AIR-CONDITIONING SYSTEM

IJAZAH : **SARJANA KEJURUTERAAN**

BIDANG: KEJURUTERAAN MEKANIKAL

Saya **ZULKURNAIN BIN HASSAN**, Sesi **2019-2023**, mengaku membenarkan tesis Sarjana ini disimpan di Perpustakaan Universiti Malaysia Sabah dengan syarat-syarat kegunaan seperti berikut: -

- 1. Tesis ini adalah hak milik Universiti Malaysia Sabah
- 2. Perpustakaan Universiti Malaysia Sabah dibenarkan membuat salinan untuk tujuan pengajian sahaja.
- 3. Perpustakaan dibenarkan membuat salinan tesis ini sebagai bahan pertukaran antara institusi pengajian tinggi.
- 4. Sila tandakan (/):



(Mengandungi maklumat yang berdarjah keselamatan atau kepentingan Malaysia seperti yang termaktub di dalam AKTA RAHSIA 1972)

(Mengandungi maklumat TERHAD yang telah ditentukan oleh organisasi/badan di mana penyelidikan dijalankan)

UNIVERSITI MALAYSIA SABAH

Disahkan Oleh,

74

ZULKURNAIN BIN HASSAN MK1911017T

Tarikh: 23 Oktober 2023

(Tandatangan Pustakawan)

ANITA BINTI ARSAD
PUSTAKAWAN KANAN
JNIVERSITI MALAYSIA SABAH

(Dr. Mohd Suffian bin Misaran @ Misran)

Penyelia Utama

DECLARATION

I hereby declare that the material in this thesis is my own except for quotations, equations, summaries, and references, which have been duly acknowledged.

23 Mac 2023





CERTIFICATION

NAME : **ZULKURNAIN BIN HASSAN**

MATRIC NO. : **MI1911017T**

TITTLE : EXPERIMENTAL STUDY ON USING WASTE HEAT FROM

SOLAR PHOTOVOLTAIC COOLING FOR LIQUID DESICCANT REGENERATION IN A SOLAR AIR-

CONDITIONING SYSTEM

DEGREE : MASTER OF ENGINEERING

FIELD : **MECHANICAL ENGINEERING**

VIVA DATE : 23 MARCH 2023

CERTIFIED BY;

Signature

1. MAIN SUPERVISOR

Dr. Mohd Suffian Bin Misaran @ Misran

2. CO-SUPERVISOR

Dr. Nancy Julius Siambun

ACKNOWLEDGEMENT

Alhamdulillah, all thanks are due to Allah, who has blessed me with the ability to

finish this research degree and who has given me the strength to conquer all

challenges. Whenever I had a query regarding my research or writing, my supervisor,

Dr. Mohd. Suffian Misran @ Misaran, was always flexible and ready to help. For this,

I would want to thank him. In all my endeavours, he supports me and points me to

the proper route. You have excelled as a teacher and a mentor to me. Your guidance

has been invaluable. In addition, I appreciate Dr. Nancy Siambun, my second

supervisor, who gave me advice, support, and enlightening comments throughout my

studies.

Additionally, without the financial assistance of Universiti Malaysia Sabah

(UMS), notably the Faculty of Mechanical Engineering (FKJ), through the research

grant project no. GA19100, this research would not have been possible. I would like

to extend my appreciation to my friends who supported me in completing this thesis,

whether through direct or indirect assistance.

A family member has been the person who has meant the most to me as I

pursued this quest. I owe my wife Norazlina Binti Musa and my parents for their

continuous support. Without them, this achievement would not have been possible.

All of the sacrifices they've made on my behalf and their prayers for me have been

my only source of sustenance thus far. They are the ultimate examples to follow.

Wassalam.

Zulkurnain Bin Hassan

23 Mac 2023

İ۷

ABSTRACT

The utilization of a desiccant cooling system presents a remarkable resolution to address forthcoming energy demand issues, all the while prioritizing human comfort through its ability to effectively control temperature and humidity levels. The typical vapour compression cooling and air conditioning systems may be replaced or supplemented by liquid desiccant cooling systems that are powered by solar photovoltaic (PV) waste heat. This study aims to investigate the utilisation of solar photovoltait cooling waste heat for liquid desiccant regeneration in a solar air conditioning system. The experimental setup incorporates accurate measuring instruments specifically designed to facilitate a wide range of precise observations during the experiment. The performance evaluation of the system was conducted using a regenerator that utilized a desiccant solution consisting of up to 40% concentrated Calcium Chloride (CaCl₂). The waste heat from the desiccant, reaching temperatures of up to 60°C, was utilized in a liquid spray towers system for regeneration. The maximum calculated output power for a solar photovoltaic (PV) system with heat recovery was 185.6 W with an efficiency of 26.54%. At 1:00 p.m. the system's maximum temperature was 70°C. The experiment's findings show that the desiccant held in the reservoir can reach a temperature of 65°C. The regenerator modifies the air's specific humidity as the solution's inlet temperature and mass flow rate increase. The rate of water evaporation increases as air flow rates and desiccant solution inlet temperatures increase.

ABSTRAK

EKPERIMEN PENGGUNAAN HABA BUANGAN DARIPADA PENYEJUKAN SOLAR FOTOVOLTAIK UNTUK PENJANAAN SEMULA CECAIR DISECCANT DALAM SISTEM PENYAMANAN UDARA SOLAR

Penggunaan sistem penyejukan bahan pengering memberikan resolusi yang baru untuk menangani isu permintaan tenaga yang akan datang, dimana ia mengutamakan keselesaan manusia melalui keupayaannya mengawal tahap suhu dan kelembapan dengan berkesan. Sistem penyejukan mampatan wap dan penyaman udara biasa boleh digantikan atau ditambah dengan sistem penyejukan bahan pengering cecair yang dikuasakan oleh haba sisa fotovoltaik suria. Kajian ini bertujuan untuk mengkaji penggunaan haba sisa penyejukan fotovoltait suria untuk penjanaan semula bahan pengering cecair dalam sistem penyaman udara solar. Persediaan ujikaji menggabungkan alat pengukur yang direka khusus untuk memudahkan pemerhatian semasa eksperimen. Penilaian prestasi sistem telah dijalanka<mark>n menggu</mark>nakan penjana semula yang menggunakan larutan pengering yang terd<mark>iri daripa</mark>da Kalsium Klorida (CaCl₂) dengan kepekatan sehingga 40%. Haba buanga<mark>n daripada ba</mark>han pengering, mencapai suhu sehingga 60°C, telah digunakan dalam sistem menara semburan cecair untuk penjanaan semula. Kuasa keluaran maksimum untuk sistem PV solar dengan pemulihan haba ialah 185.6 W dengan kecekapan 26.54 peratus. Pada jam 1.00 petang, suhu maksimum sistem ialah 70 °C. Dapatan eksperimen menunjukkan bahan pengering di dalam takungan boleh mencapai suhu 65°C. Penjana semula mengubah suai kelembapan khusus udara apabila suhu masuk larutan dan kadar aliran jisim meningkat. Kadar penyejatan air meningkat apabila kadar aliran udara dan suhu masuk larutan bahan pengering meningkat

LIST OF CONTENTS

			Page
TITL	E		i
DECL	ARATIO)N	ii
CERT	TIFICAT	ON	iii
ACKI	NOWLED	GEMENT	iv
ABST	TRACT		٧
ABS	TRAK		vi
LIST	OF CON	TENTS	vii
LIST	OF TAB	LES	Xi
LIST	OF FIGU	JRES	Xii
LIST	OF ABB	REVIATIONS	XV
LIST	OF APP	ENDICES	xviii
15-17	PTER 1:	INTRODUCTION	
1.1 1.2	Introdu Backgr	ound of the Study	1 1
1.3	Probler	Statement UNIVERSITI MALAYSIA SABAH	4
1.4	Resear	ch Objectives	5
1.5	Resear	ch Questions	5
1.6	Signific	cance of the Study	6
1.7	Scope a	and Limitations of Research	6
1.8	Potenti	al Benefits	7
1.9	Structu	re of the Thesis	8
СНАІ	PTER 2:	LITERATURE REVIEW	
2.1	Introdu	ıction	10
2.2	Heating	g, Ventilation and Air Conditioning (HVAC)	12
	2.2.1	Conventional Vapor Compression Refrigeration System	13

	2.2.2	Solar Air Conditioning	14
	2.2.3	Liquid desiccant Air Conditioning (LDAC)	18
	2.2.4	Liquid Desiccant Evaporative Cooler (LDEC)	19
2.3	Fundar	nentals of Desiccant Cooling	21
	2.3.1	Desiccant Dehumidifier	21
	2.3.2	Hot Source	22
2.4	System	for Liquid Desiccant Cooling (LDCS)	23
	2.4.1	Dehumidification Device	25
	2.4.2	Dehumidifier System Type	26
2.5	Liquid	Desiccant Materials	30
	2.5.1	Calcium Chloride as Liquid Desiccants	32
2.6	Packag	ing Materials	34
	2.6.1	Structured Packing	34
A	2.6.2	Random Packing	36
B	2.6.3	Spray Packaging	38
6	2.6.4	Packing of the Falling Film Type	39
2.7	Experir	nent Studies on Liquid Desiccant Cooling Systems	41
2.8	Review	On Solar PV/T for Heating Applications	44
2.9	Gap Ar	alysis	48
2.10	Summa	ary	49
CHAP	TER 3:	METHODOLOGY	
3.1	Introdu	iction	50
3.2	Prepara	ation of Calcium Chloride	51
3.3	Liquid I	Desiccant Heating Via Solar PV-T	52
	3.3.1	Solar PV with Thermal Recovery (Solar PV-TR) System Description	53
	3.3.2	Solar PV-T Test Procedure and Data Collection	55
3.4	Perforn System	nance of Liquid Desiccant Regenerator and Humudifier	57

	3.4.1	Description of the Liquid Desiccant Regenerator and Humidifier System	57
	3.4.2	Regenerator and Humidifier Test Procedure and Data Collection	64
3.5	Perforn	nance of Evaporative Cooler System	67
	3.5.1	Description of the Liquid Desiccant and Direct Evaporative Cooling System	67
	3.5.2	Solar Liquid Desiccant Evaporative Cooler and Direct Evaporative Cooler test procedure and data collection	70
3.6	Summa	ary	72
CHAP	PTER 4:	RESULTS & DISCUSSIONS	
4.1	Introdu	uction	73
4.2	Liquid I	Desiccant Heating and Solar PV Performance	73
	4.2.1	Liquid Desiccant Heating	75
165	4.2.2	Solar PV Thermal and Electrical Performance	77
B	4.2.3	Summary	79
4.3	Dehum	idifier and Regenerator Performance	80
	4.3.1	Regenerator	81
	4.3.2	Dehumidifier UNIVERSITI MALAYSIA SABAH	89
4.4	Solar L	iquid Desiccant Evaporative Cooler Performance	98
4.5	Summa	ary	102
CHAP	PTER 5:	CONCLUSION & RECOMMENDATION	
5.1	Conclus	sion	104
5.4	Future	Scope	110
REFE	RENCES		112
APPE	APPENDICES 1		

LIST OF TABLES

			Page
Table 2.1	:	Solar Absorption and Adsorption Chillers, Solar Ejector Cooling, And Solar Deicing (Liquid and Solid) Cooling	16
Table 2.2	:	Comparison of Diverse Solar Thermal Cooling	18
Table 3.1	:	CaCl ₂ .2H ₂ O is used to produce calcium chloride solutions of varying concentrations and volumes	52
Table 3.2	:	Solar PV Waste Heat Recovery System Component Specification	53
Table 3.3	:	The Specification of the Solar PV Module	55
Table 3.4	:	Measuring Instruments for Solar Panel	57
Table 3.5	:	Specifications of Components Used in Dehumidifier	58
Table 3.6	:	Specifications of Components Used in Regenerator	59
Table 3.7	:	Specifications of Celdek Packing	60
Table 3.8	-	Hydrated Calcium Chloride (Dow, 2003)	61
Table 3.9		Calcium Chloride Solution Properties at 25°C	62
Table 3.10		Specifications of Pump	63
Table 3.11		Pump Specification	63
Table 3.12	100	Specifications of Axial Fan	64
Table 3.13	A	Measuring Instruments for Liquid Desiccant Air A B A B Conditioning	66
Table 3.14	:	Evaporative Cooler Design and Specification	68
Table 3.15		Specifications of Axial Fan	70
Table 3.16	:	Measuring Instruments for Evaporative Cooler	72
Table 4.1	:	Solar PV with and without Heat Recovery Performance	75
Table 4.2	:	Value of The Experimental Test for Average, Maximum and Minimum	99

LIST OF FIGURES

			Page
Figure 1.1	:	Desiccant Cooling Principle	4
Figure 1.2	:	Hybrid Liquid Desiccant Systems Classification	7
Figure 1.3	:	Basic Concept of The Research Project	7
Figure 2.1	:	Desiccant Cooling Principle	22
Figure 2.2	•	Diagrammatic Representation of Liquid Desiccant Cooling System	24
Figure 2.3	:	Vapour Pressure Variations in The Desiccant Cooling System	25
Figure 2.4	:	Solid Desiccant Cooling System	27
Figure 2.5	:	Systems for Liquid Desiccant Cooling	28
Figure 2.6	:	Hybrid Desiccant Cooling System	29
Figure 2.7		Solar Desiccant Air Conditioning	30
Figure 3.1	40	Schematic Drawing of the Experimental Setup	51
Figure 3.2		Actual Vies of Experimental Setup	51
Figure 3.3	D	Solar Panel with Cooper Pipe System	54
Figure 3.4	and)	The Front View of The PVT Collector	54
Figure 3.5	Y.B	Panel PVT Waste Heat Recovery System A SABAL	54
Figure 3.6	:	The schematic diagram and the experimental setup for Solar Panel PVT	55
Figure 3.7		Dehumidifier	58
Figure 3.8	:	Regenerator	59
Figure 3.9	:	Celdek Structured Packing	60
Figure 3.10	:	Calcium Chloride	62
Figure 3.11	:	Water Pump	63
Figure 3.12	:	Axial Fan	64
Figure 3.13	:	A Direct Evaporative Cooling And Its Pad	68
Figure 3.14	:	Body of Evaporative Cooler	68
Figure 3.15	:	Honeycomb Pad	69

Figure 3.3	16 :	Suction Fan	69
Figure 4.1	l :	Hourly Average Solar Radiation	74
Figure 4.2	2 :	Solar PV Top Surface and Back Surface Temperature for System with and without heat recovery	76
Figure 4.3	3 :	Solution Outflow into And Out of The Tank	77
Figure 4.4	1 :	Thermal Efficiency of Solar PV System	78
Figure 4.5	5 :	Power Efficiency of Solar PV System	78
Figure 4.6	5 :	Power Output of Solar PV System	79
Figure 4.7	7 :	The Relationship Between Air Flow Rate and Air Specific Humidity	82
Figure 4.8	3 :	The Relationship Between Air Flow Rate and Evaporation Rate	82
Figure 4.9) :	The Relationship Between Air Flow Rate and Mass Transfer Coefficient	83
Figure 4.1	10 :	The Relationship Between Air Flow Rate and Outlet Air Temperature	83
Figure 4.1	11 :	The Relationship Between Air Flow Rate and Outlet Solution Temperature	84
Figure 4.1	12 :	The Relationship Between Air Flow Rate and Effectiveness of the Regenerator	84
Figure 4.1	13 :	The Relationship Between Inlet Desiccant Solution Concentration and Specific Humidity Of The Air	86
Figure 4.1	14 :	The Relationship Between Inlet Desiccant Solution Concentration and Evaporation Rate	86
Figure 4.1	15 :	The Relationship Between Inlet Desiccant Solution Concentration and Mass Transfer Coefficient	87
Figure 4.1	16 :	The Relationship Between Inlet Desiccant Solution Concentration and Outlet Air Temperature	87
Figure 4.1	17 :	The Relationship Between Inlet Desiccant Solution Concentration and Outlet Solution Temperature	88
Figure 4.1	18 :	The Relationship Between Inlet Desiccant Solution Concentration and Effectiveness Of The Regenerator	88

Figure 4.19	•	The Relationship Between Inlet Solution Temperature and Specific Humidity	90
Figure 4.20	•	The Relationship Between Inlet Solution Temperature and Air Outlet Temperature	91
Figure 4.21	:	The Relationship Between Inlet Solution Temperature and Solution Outlet Temperature	91
Figure 4.22	:	The Relationship Between Inlet Solution Temperature and Moisture removal rate	92
Figure 4.23	:	The Relationship Between Inlet Solution Temperature and Mass Transfer Coefficient	92
Figure 4.24	:	The Relationship Between Inlet Solution Temperature and Effectiveness of the Dehumidifier	93
Figure 4.25	:	The Relationship Between Desiccant Solution Concentration and Specific Humidity of the Air	94
Figure 4.26	: ====================================	The Relationship Between Desiccant Solution Concentration and Air Outlet Temperature	95
Figure 4.27		The Relationship Between Desiccant Solution Concentration and Solution Outlet Temperature	96
Figure 4.28		The Relationship Between Desiccant Solution Concentration and Moisture Removal Rate	96
Figure 4.29	N.	The Relationship Between Desiccant Solution Concentration and Mass Transfer Coefficient	97
Figure 4.30	:	The Relationship Between Desiccant Solution Concentration and Effectiveness of the Dehumidifier	98
Figure 4.31	:	Dry-Bulb Inlet, Dry-Bulb Outlet And Wet-Bulb Inlet With Time For Liquid Desiccant Evaporative Cooler and Direct Evaporative Cooler	100
Figure 4.32	:	Variation Of Cooling Capacity With Time	101
Figure 4.33	:	Performance Of Saturation Efficiency With The Time	101
Figure 4.34	•	Variation Of Feasibility Index With Time	102

LIST OF SYMBOL

nel-Electrical efficiencynth-Thermal efficiencyApvt-Collector area [m²]

εdeh - Dehumidifier effectiveness

Fw - Fraction of total area available for mass

Transfer

Fh - Fraction of total area available for heat

Transfer

G - Irradiance [W/m²]

Current of PV Module at max. Power
 Voltage of PV Module at Max. Power.
 Outlet Water Temperature [°C]

Ti - Collector Inlet Water Temperature [°C]

m - Mass Flow Rate [kg/s]

Cp - Specific Heat of Water [J/kg K]

Qc - Cooling Capacity [kJ/h]ma - Air Flow Rate [kg/s]

Cpa - Specific Heat of Air [J/kgK] **Twb** - Wet bulb temperature, °C

wa - Humidity ratio of inlet air [g/kg of dry air]

ma - Mass flow rate of inlet air [kg/s]

T Temperature
F* Feasibility index

Pd - Partial pressure of dry air [kpa]

Pv - Partial pressure of water vapour [kpa]
Psat - Partial pressure of saturated air [kpa]

ha -UN Enthalpy of inlet air 151A 5ABAT **Ha** - specific enthalpy

ms - mass flow rate of the air

ma - Mass flow rate of desiccant solution

Cs - desiccant concentration

Hs - Specific enthalpy

Tsi - Temperature of solution inlet **SE** - Saturation effectiveness

L - Length

n - Number of steps
Iroc - Current at ROC (A)
Istc - Current at STC (A)
Vroc - Voltage at ROC (V)
Vstc - Voltage at STC (V)
FF - Fill Factor (decimal)
Voc - open circuit voltage (V)

Vmp - voltage at maximum power (V)

Isc - short circuit current (A)

Imp-current at maximum power (A)Tcell-cell or module temperature ρ -Density of the air [kg/m³]

Effectiveness [Dimensionless] ε Specific humidity [kg/kg of dry air]
Change in specific humidity [kg/kg of dry air] ω dω Uncertainty [Dimensionless] ξ Relative humidity φ Specific humidity ω_{o} Velocity of air (m/s) v Dry air a Inlet of air ai Outlet of air ao Air to solution as Average av Atmospheric atm Dehumidifier d Equilibrium eq evap Evaporation rate Inlet max Outlet 0 Partial pressure p Regenerator Solution S si Inlet of solution Outlet of solution SO

UNIVERSITI MALAYSIA SABAH

LIST OF ABBREVIATIONS

A - Ampere

CaCl₂ - Calcium chloride CFC - Chlorofluorocarbon

CPVC - Coefficient of Performance
- Chlorinated polyvinyl chloride
- Cost-Effective Liquid Desiccant

DBT - Dry bulb temperatureDEG - Diethylene Glycol

H2O/KCOOH - potassium

HVAC - heating, ventilation, and air-conditioning

I - Current

IAQ - Indoor air quality

KCOOH - Non-Corrosive Potassium

LDAC - Liquid desiccant air conditioning
LDCS - Liquid Desiccant Cooling System]
LDD - Liquid Desiccant Dehumidification

LIQUID desiccant dehumidification cooling
LIQUID LIQUID Desiccant Evaporative Cooler

LiBr - Lithium Bromide
LiCl - Lithium chloride
LPH - Litres per hour
MEG - Monoethylene-Glycol
MRR - Moisture Removal Rate
NTUm - Nephelometric Turbidity

P - Power

PV - UNI Photovoltaics ALAYSIA SABAH

PV/T - Photovoltaics/Thermal PVC - Polyvinyl Chloride

SIE - Solar-Driven Interfacial Evaporation

TEG - Triethylene glycol
 TR - Ton of refrigeration
 UMS - University Malaysia Sabah

V - Voltage

VCS - Vapor Compression System

W - Watt

WBT - Wet bulb temperature

LIST OF APPENDICES

		Page
Appendix A	: Solar Irradiance Data	128
Appendix B	: Solar PV Performance with Heat Recovered	128
Appendix C	: PV Performance without Heat Recovered	129
Appendix D	: Flow Rate's Affect on Regenerator Outlet	129
	Parameters	
Appendix E	: Effect of Inlet Desiccant Solution Concentratio	n 130
	on Regenerator Outlet Parameters	
Appendix F	: The Effect of Inlet Solution Temperature on	130
	Dehumidifier Outlet Parameters	
Appendix G	: The Effect of Desiccant Solution Concentration	on 131
	Dehumidifier Outlet Parameters	
Appendix H	: Performence Solar Liquid Desiccant Evaporativ	ve 131
691	Cooler	
Appendix I	: Performence Direct Evaporative Cooler	132
a		
151		

UNIVERSITI MALAYSIA SABAH

CHAPTER 1

INTRODUCTION

1.1 Introduction

This chapter presents an introduction to the research topic, which is focused on the experimental investigation of utilizing waste heat from solar PV cooling for the regeneration of desiccant in a liquid desiccant evaporative cooler. The chapter provides background information on the reasons behind selecting this research title and how the problem arose. It discusses the problem statements, aims, parameters, and potential benefits of the study. Additionally, the chapter outlines the study plan, which contains crucial information that will influence the progression of this project.

1.2 Background of the Study VERSITI MALAYSIA SABAH

Air conditioning plays a crucial role in managing both sensible and latent cooling loads, especially in hot and humid regions. The primary considerations for designing air conditioning systems encompass factors such as thermal comfort, indoor air quality, energy efficiency, and their environmental impact, as noted by Vakiloroaya et al. in 2014. The commonly employed method is mechanical vapor compression, which entails cooling air to its dew point temperature to extract moisture. However, it's important to note that additional energy is required to reheat the air to the desired supply temperature, as pointed out by Xiao et al. in 2016.

Due to rising cooling loads and comfort demands, tropical countries like Malaysia are seeing an increase in the need for cooling indoor air. Air conditioning has become an indispensable cooling mechanism in all office buildings throughout Malaysia, meeting the essential need for indoor comfort. However, it is important to note that air conditioning systems are the primary consumers of energy, accounting for approximately 40% of the total electricity consumption in office buildings, exceeding the energy consumption of lighting and other electrical appliances. Moreover, small office buildings, including shop-office buildings, represent a significant portion, comprising around 57% of the total office building inventory in Malaysia (Sopian et al., 2009). Conventional cooling methods were used by most small office buildings, often with an electrically driven compressor system that has several obvious drawbacks, including high energy consumption and high electricity peak demand. Typically, it uses refrigerants that have a harmful influence on the environment. On the other hand, mechanical vapour compression systems have several related risks that have a significant effect on indoor air quality and occupant health. These issues include water condensation on the cooling coil surface, bacterial growth, and fungus. To address these issues, several cutting-edge dehumidification cooling systems with efficient independent temperature and humidity controls and concurrently lower energy usage have been developed.

During extremely hot and humid conditions, the temperature tends to reach or hover around its maximum point. This typically occurs when the intensity of solar radiation is at its peak. The variable supply of solar energy and the demand for air conditioning are interconnected. Due to the correlation between the intensity of solar radiation and the load on air conditioning systems, there exists a potential to reduce energy consumption in air conditioning by harnessing solar energy.

Ongoing research into solar cooling technologies presents a promising avenue for mitigating the challenges posed by the peak power demand and energy consumption typically linked to conventional air conditioning systems. These innovations aim to address the substantial issues encountered with compression cooling systems, as highlighted in the 2016 report by the International Energy Agency Solar Heating and Cooling Program. Solar cooling, which utilizes solar energy to cool indoor spaces, stands out as a notable alternative to traditional vapor compression-cycle HVAC systems. Liquid desiccant air conditioning, in particular, is gaining increasing attention as a potential replacement.

The liquid desiccant dehumidification system, or LDDC system, is a hybrid air conditioning system that combines a cooling unit to manage the sensible cooling load with a liquid desiccant dehumidification unit to handle the latent cooling load (Mohammad et al, 2013). Desiccant dehumidification may be performed using solid or liquid desiccant systems (Salikandi et al, 2021). Liquid desiccant dehumidification (LDD) offers more benefits than solid desiccant dehumidification because it has a lower mass transfer resistance, a greater moisture absorption capacity, a lower airflow pressure drops, and a smaller temperature range for desiccant regeneration (SDD) (Oh et al., 2017).

A few cooling methods that can be integrated with the LDD system are evaporative cooling, vapour compression cooling, and absorption cooling. System energy efficiency can be greatly increased by using hybrid systems. (Yamaguchi et al., 2011). Due to its less expensive setup and maintenance (Heidarinejad et al., 2009), evaporative cooling has been widely used. An evaporative cooler uses about 75% less energy than a conventional cooler. (Cerci, 2003).

Liquid desiccant evaporative cooling systems integrate a liquid desiccant dehumidification system with an evaporative cooling system to enhance overall system efficiency and make use of solar energy resources. In such a hybrid configuration, the evaporative cooling system handles the sensible heat load, while the desiccant dehumidification system is responsible for removing the latent load, as detailed in Yong et al.'s work from 2006. Furthermore, because the regeneration phase of the desiccant dehumidification system only requires low-grade energy, it becomes feasible to power liquid desiccant air conditioning systems with renewable energy sources like solar energy, as discussed in Cheng et al.'s research from 2013. For this purpose, we can use the technique of utilizing waste heat from solar PV. Solar Panel waste heat refers to heat that is generated in solar panel processes and then rejected to the environment even though it could be utilized for some users. Therefore, reusing and recovering waste heat is a practical technique to significantly increase energy efficiency. Solar energy is also a clean and renewable source of energy and can generate low-grade heat.

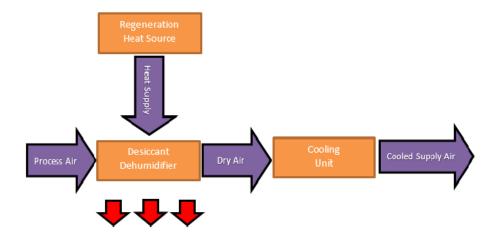


Figure 1.1 : Desiccant Cooling Principle

1.3 Problem Statement

The increasing global use of air conditioning systems necessitates improvements to address challenges related to high energy consumption and peak power demands. It is crucial to explore alternatives to chemical refrigerants to minimize the environmental impact of air conditioning systems. The development and promotion of environmentally friendly systems have become essential due to the worldwide significance of energy and environmental concerns. Renewable and waste heat sources, such as solar heat, waste steam, flue gases, and used cooling water, offer viable options as alternatives to traditional thermal or electrical energy sources due to their higher exergy and cost-effectiveness.

Evaporative cooling emerges as an environmentally friendly technology as it utilizes water as a working substance, eliminating the need for CFCs and reducing greenhouse gas emissions. Operating this technology is relatively straightforward. Evaporative cooling occurs when a liquid in contact with an object cool through evaporation, typically into the surrounding air. The cooling effect is maximized when dry air passes over a wet surface, resulting in increased evaporation and greater cooling. However, the effectiveness of evaporative cooling is reduced in hot climates with high atmospheric moisture content.

1.4 Research Objectives

This study aims to investigate the ability of heat recovered from cooling a solar PV to assist in regeneration of liquid desiccant for a Liquid Desiccant Evaporative Cooler (LDEC). The objectives of the study are:

- To measure the maximum temperature achievable by cooling the solar PV surface, and how does this temperature impact the electricity generation efficiency of the system in terms of its performance.
- ii. To evaluate the impact of various operational parameters, such as air flow rate, inlet desiccant solution concentration and inlet solution temperature on the efficiency of the regenerator and dehumidifier when utilizing waste heat recovered from solar PV cooling.
- iii. To compare the performance of the LDEC, utilizing waste heat from solar PV cooling, with a traditional Direct Evaporative Cooler (DEC) in terms of cooling efficiency, temperature difference between inlet and outlet air, and humidity change.

1.5 Research Questions

The following main research questions must be answered in order to evaluate the system's effectiveness:

UNIVERSITI MALAYSIA SABAH

- RQ1 What is the maximum solution temperature achievable by cooling the solar PV surface, and how does this temperature impact the electricity generation efficiency of the LDEC system in terms of its performance?
- RQ2 What is the impact of various operational parameters, such as air flow rate, inlet desiccant solution concentration and inlet solution temperature on the efficiency of the regenerator and dehumidifier when utilizing waste heat recovered from solar PV cooling?
- RQ3 How does the performance of the Liquid Desiccant Evaporative Cooler (LDEC), utilizing waste heat from solar PV cooling, with a traditional Direct Evaporative Cooler (DEC) in terms of cooling efficiency, temperature difference between